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Double Slider Crank Chain

A kinematic chain which consists of two turning pairs and two sliding pairs is known as **double slider crank chain**, as shown in Fig. We see that the link 2 and link 1 form one turning pair and link 2 and link 3 form the second turning pair. The link 3 and link 4 form one sliding pair and link 1 and link 4 form the second sliding pair.

Inversions of Double Slider Crank Chain

The following three inversions of a double slider crank chain are important from the subject point of view:

1. Elliptical trammels. It is an instrument used for drawing ellipses. This inversion is obtained by fixing the slotted plate (link 4), as shown in Fig. The fixed plate or link 4 has two straight grooves cut in it, at right angles to each other. The link 1 and link 3, are known as sliders and form sliding pairs with link 4. The link AB (link 2) is a bar which forms turning pair with links 1 and 3. When the links 1 and 3 slide along their respective grooves, any point on the link 2 such as P traces out an ellipse on the surface of link 4, as shown in Fig (a). A little consideration will show that AP and BP are the semi-major axis and semi-minor axis of the ellipse respectively. This can be proved as follows :

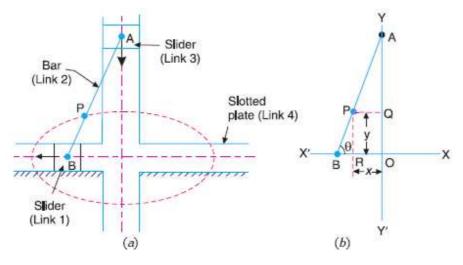


Fig: Elliptical trammels.

Let us take OX and OY as horizontal and vertical axes and let the link BA is inclined at an angle θ with the horizontal, as shown in Fig (b). Now the co-ordinates of the point P on the link BA will be

$$X = PQ = AP \cos \theta$$
; and $y = PR = BP \sin \theta$

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Or

$$\frac{x}{AP} = \cos \theta$$
; and $\frac{y}{BP} = \sin \theta$

Squaring and adding

$$\frac{x^{2}}{(AP)^{2}} + \frac{y^{2}}{(BP)^{2}} = \cos^{2}\theta + \sin^{2}\theta = 1$$

This is the equation of an ellipse. Hence the path traced by point P is an ellipse whose semi-major axis is AP and semi-minor axis is BP.

Note: If P is the mid-point of link BA, then AP = BP. The above equation can be written as

$$\frac{x^2}{(AP)^2} + \frac{y^2}{(AP)^2} = 1 \quad \text{or} \quad x^2 + y^2 = AP^2$$

This is the equation of a circle whose radius is AP. Hence if P is the mid-point of link BA, it will trace a circle.

2. Scotch yoke mechanism. This mechanism is used for converting rotary motion into a reciprocating motion. The inversion is obtained by fixing either the link 1 or link 3. In Fig, link 1 is fixed. In this mechanism, when the link 2 (which corresponds to crank) rotates about B as centre, the link 4 (which corresponds to a frame) reciprocates. The fixed link 1 guides the frame.

3. Oldham's coupling. An oldham's coupling is used for connecting two parallel shafts whose axes are at a small distance apart. The shafts are coupled in such a way that if one shaft rotates, the other shaft also rotates at the same speed. This inversion is obtained by fixing the link 2, as shown in Fig (a). The shafts to be connected have two flanges (link 1 and link 3) rigidly fastened at their ends by forging.

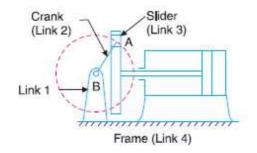
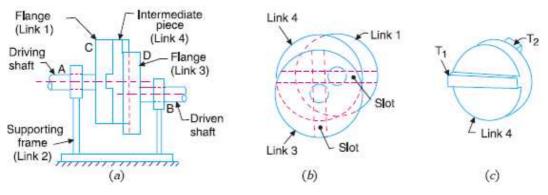


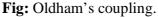
Fig: Scotch yoke mechanism

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The link 1 and link 3 form turning pairs with link 2. These flanges have diametrical slots cut in their inner faces, as shown in Fig (b). The intermediate piece (link 4) which is a circular disc, have two tongues (i.e. diametrical projections) T_1 and T_2 on each face at right angles to each other, as shown in Fig (c). The tongues on the link 4 closely fit into the slots in the two flanges (link 1 and link 3). The link 4 can slide or reciprocate in the slots in the flanges.





When the driving shaft A is rotated, the flange C (link 1) causes the intermediate piece (link 4) to rotate at the same angle through which the flange has rotated, and it further rotates the flange D (link 3) at the same angle and thus the shaft B rotates. Hence links 1, 3 and 4 have the same angular velocity at every instant. A little consideration will show, that there is a sliding motion between the link 4 and each of the other links 1 and 3. If the distance between the axes of the shafts is constant, the centre of intermediate piece will describe a circle of radius equal to the distance between the axes of the two shafts. Therefore, the maximum sliding speed of each tongue along its slot is equal to the peripheral velocity of the centre of the disc along its circular path.

Let ω = Angular velocity of each shaft in rad/s, and

r = Distance between the axes of the shafts in metres.

 \Box Maximum sliding speed of each tongue (in m/s),

 $v = \omega . r$

Reference: Theory of Machines - R. S. Khurmi & J.K.Gupta

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Peaucellier mechanism

1. Peaucellier mechanism. It consists of a fixed link OO_1 and the other straight links O_1A , OC, OD, AD, DB, BC and CA are connected by turning pairs at their intersections, as shown in Fig. The pin at A is constrained to move along the circumference of a circle with the fixed diameter OP, by means of the link O_1A . In Fig,

$$AC = CB = BD = DA$$
; $OC = OD$; and $OO_1 = O_1A$

It may be proved that the product $OA \times OB$ remains constant, when the link O_1A rotates. Join CD to bisect AB at R.

Now from right angled triangles ORC and BRC, we have

 $OC_2 = OR_2 + RC_2$...(i) and $BC_2 = RB_2 + RC_2$...(ii)

Subtracting equation (ii) from (i), we have

$$OC_2 - BC_2 = OR_2 - RB_2$$

= (OR + RB) (OR - RB)
= OB × OA

Since OC and BC are of constant length, therefore the product $OB \times OA$ remains constant. Hence the point B traces a straight path perpendicular to the diameter OP.

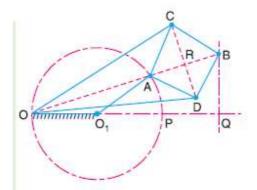


Fig: Peaucellier mechanism

Reference: Theory of Machines - R. S. Khurmi & J.K.Gupta